

Date of publication xxxx 00, 2022, date of current version xxxx 00, 2022.

Digital Object Identifier 10.1109/ACCESS.2017.DOI

Design Aspects and Thermal Characteristics of Single-Sided Linear Induction Motor for Electromagnetic Launch Application

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ABSTRACT This paper presents the design aspects and thermal characteristics of single-sided linear induction motor (SLIM) for Electromagnetic launchers. It is designed to accelerate a 50 kg mass through a distance of 3 meters within a minuscule time to an exit velocity of 50 m/s. A study of the motor's parametric variation based on the launching requirement has been performed. Its performance characteristics are obtained using 3D FEM and verified analytically using Parseval's method. The FEM, analytical and experimental verification of the thrust characteristics of different secondary conductive sheet materials namely, aluminum, beryllium copper and German silver shows that, the latter exhibits superior characteristics than the former two, because its thrust-velocity curve is close to inverse linear relationship (stable region of operation of LIM), which significantly reduces the accelerating time during the launch. The designed SLIM is assessed for its thermal performance with materials having temperature-dependent electrical properties by carrying out a coupled-field FEM simulation. The conclusions demonstrate that the electromagnetic and thermal assessments are in tandem with each other as required for launching.

INDEX TERMS Electromagnetic launcher, SLIM, Fourier Transform Model, Coupled-field FEM.

I. INTRODUCTION

THE existing launching methods have reached a saturation point where they cannot meet modern launcher requirements such as higher projectile velocity and specific thrust. Comparatively, the launcher costs and simplification of the equipment drastically improves with linear electromagnetic (EM) launcher, which enables an object to be propelled within a minuscule time attaining a certain high velocity using EM force [1], [2]. It finds its application in national defense, traffic and aerospace owing to its advantages which include simple structure, recyclability, controllability, etc [3], [4]. It is reported to accelerate a cannonball to a very high velocity [5]. EM catapults can also be applied to propel unmanned vehicles, tactical missiles, aircrafts and fish torpedoes [6]. Electromagnetic launchers are in use for launching drones having lesser capability for self-take off.

The use of launching technologies for space exploration has been reviewed and indicates a strong correlation with

defense applications and rocket launcher systems [7], [8]. By employing the principles of railgun and coilgun, higher muzzle velocity can be achieved [9]. Due to the serious limitation of burning of the tracks as a result of arcing at the sliding contacts of such systems, several other topologies, including the tubular type induction launcher, were investigated, whose maximum speed is limited by current-collection problems [10]. LIM is used for comparatively lower velocity for safe launching of mass. Linear permanent-magnet machines have also gained popularity for EM launch as they overcome the low efficiency problem present in linear induction motors (LIMs), and have been successfully developed for vertical launch and EM aircraft launch (EMAL) applications [11], [12]. Once considered insurmountable, the technical barriers have yielded to focused theoretical, computational, and experimental research [13] and several countries all over the world have taken up the challenge of developing and understanding the art and science of EM launch and launchers

[14]–[17].

Long-primary single-sided LIMs (SLIMs) offer higher force density and acceleration due to its secondary conductive sheet (mover) being lightweight as compared to other types of LIMs which is suited for EM launch application. Also, there is no requirement of sliding contacts for current collection. Additionally, when combined with high temperature superconducting magnetic levitation system, long-primary SLIM would operate without any friction loss and guidance control system [18].

The current work focuses on designing an SLIM to accelerate a mass to an exit velocity and travel a distance within a short time. A mathematical model of the EM launch system (EMLS) is developed in section II. Section III pertains to the designing of SLIM based on a study of its parametric variation. Its performance is predicted using 3D FEM MAXWELL Software and validated analytically by Parseval's method for constant current drive [22] given in the Appendix. Section IV demonstrates, how the temperature distribution of the motor is also determined with materials having temperature-dependent electrical properties by carrying out a coupled-field simulation. The results are compared with those obtained by an analytical model in section V. The designed SLIM satisfies the performance specifications. Finally in section VI the key challenges involved and their possible solutions are also highlighted by the authors.

A fair comparison of the proposed work is being made with few select published works in literature involving the electromagnetic as well as thermal assessment of EMALs in Table 1.

II. MATHEMATICAL MODEL

For simplifying the modeling of the EMLS, consider a body of mass m placed on a plane inclined at an angle θ with the horizontal Earth's surface. The electromagnetic force F (thrust) generated by the SLIM along the inclined plane is continuously applied to the body (mover secondary of the SLIM and the payload attached to it) and it moves with a velocity v .

Then the equation of motion becomes,

$$F - m \frac{dv}{dt} = mg \sin\theta + \mu (mg \cos\theta - F_p) + b v^2 \quad (1)$$

where μ is the coefficient of kinetic friction, b is the coefficient of drag force and F_p is the normal direction electromagnetic force generated by the SLIM. It will become evident in section III B from Fig. 14b, that the normal force generated by the designed SLIM is repulsive in the region of interest i.e. between standstill and a little beyond 50 m/s (exit velocity of the projectile). The secondary mover of the SLIM is considered to be placed above the primary stator. The generated normal force will levitate the mover which is controlled by friction to overcome the normal and lateral displacements. In EM launch applications, tremendous amount of electrical energy flows in the stator coils of the LIM for a short duration to provide maximum acceleration to the projectile. To avoid

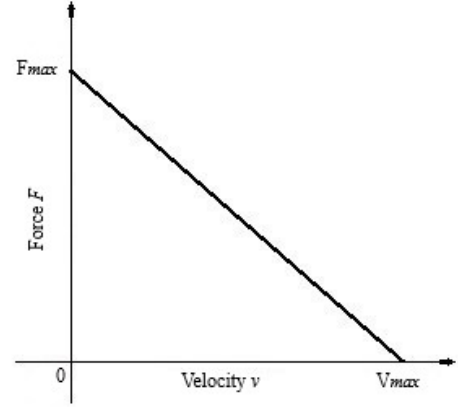


FIGURE 1: Assumed inverse linear force-velocity characteristic of the LIM

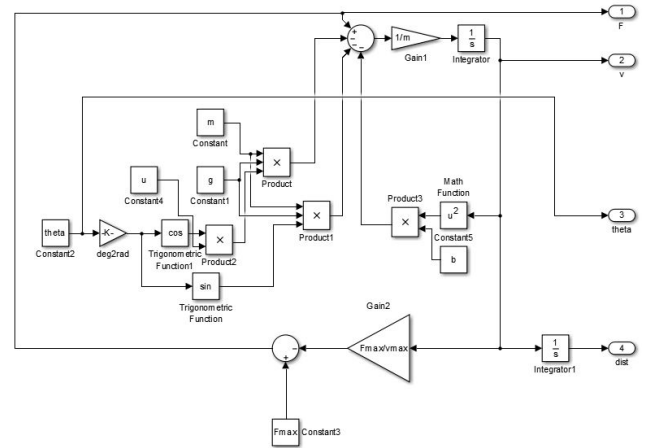


FIGURE 2: Simulation model used for predicting SLIM performance while operating

insulation breakdown in the coils due to the heat generated in them, the operating time of the launcher should be kept as low as possible, while achieving the desired performance. For this purpose, the characteristic between the applied force, which would be generated by the LIM stator, and the velocity must be as close to an inverse linear relationship as possible. By avoiding the positive slope region in the usual force-velocity characteristic of an LIM, the starting (accelerating) time of the motor reduces significantly. Another option is to provide a constant thrust by V/F method for launch application wherein the thrust requirement is almost half the value of peak thrust requirement in inverse linear characteristics. Though inverse linear characteristic results in quick launch. Thus, the following relation between the force F and the velocity v is assumed:

$$F = F_{max} - \frac{F_{max}}{v_{max}} v \quad (2)$$

where F_{max} and v_{max} are the maximum limit of force and velocity starting from 0 (Fig. 1).

TABLE 1: Comparison of the proposed work with published references

Reference	Type of machine	EMAL	Electromagnetic Thrust Assessment			Thermal Assessment		
			Analytic	FEM	Experimental	Analytic	FEM	Experimental
Superconducting Technology [19]	SCLSM	Civil	Multilayer theory (MLT)	Iterative	✗	Modeling of AC losses	Iterative	✗
Different pole Arcs [20]	PMLSM	Defence	✗	✓	✗	Transient heat conduction	Primary core only	Only Winding temperature
Different m/c topologies [21]	LIM and PMLSM	Civil	Equivalent circuit model + (MLT)	Iterative	✗	✗	✓ + FDM	✗
Proposed work	LIM	Defence	Parseval's	✓	Partial	Lumped parameter mode for complete machine	Coupled EM/Mech.	Proposed for future work

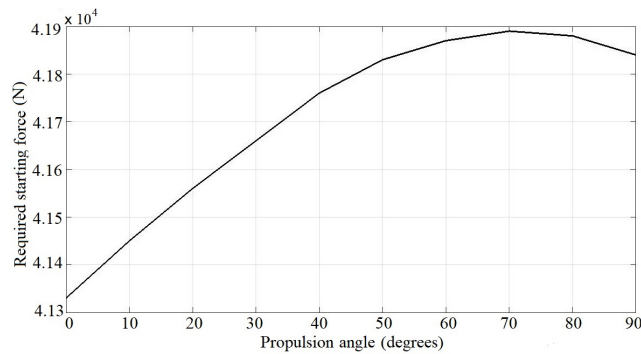


FIGURE 3: Required values of starting force for different values of launch angle

Equations (1) and (2) describe the accelerated motion of the body up the inclined plane. These two equations were setup in MATLAB Simulink (Fig. 2). Also, a script was written in MATLAB to find the required value of starting force F_{max} to satisfy the launcher requirements.

SIMULATION AND RESULTS

The following parameters (Table 2) were set for the simulation model with an aim to accelerate a body of mass 50 kg at rest to attain a velocity of 50 m/s while traveling a distance of 3 meters. With the above objective, the required values of F_{max} for different values of θ were found which would satisfy the performance requirements.

TABLE 2: MATLAB Simulink model parameters

Parameter	Symbol	Value	Unit
Total mass of body	m	50	kg
Acceleration due to gravity	g	9.81	m/s ²
Launch angle	θ	0 - 90	degrees
Coefficient of drag force	b	0.06	N/m ² /s ²
Coefficient of kinetic friction	μ	0.35	-
Maximum velocity	v_{max}	75	m/s

Fig. 3 shows that horizontal launch of the projectile would require the least but still a considerable amount of starting force of around 41.35 kN. Also, the starting force value does not vary much (about 1.33% as compared to horizontal launch case) when the launch angle is varied. It means that the force-velocity characteristics and hence the LIM design would remain more or less the same for any launch angle between 0° and 90°. The time to achieve the desired exit ve-

locity can also be found, which would be crucial in deciding the operating time of the LIM and its temperature rise. Also, the power and energy required at the time of launching can be determined by using the force, velocity and time sample values. The results are shown for a launch angle of 30°, with the rest of the parameters as indicated in Table 2.

Fig. 4a shows the force-time and the inverse linear force-velocity characteristic. With a starting force of around 42 kN, it takes 0.101 seconds for the projectile to reach the exit velocity of 50 m/s within the stipulated travel distance of 3 meters. The synchronous or maximum velocity of the LIM is chosen as 75 m/s, but the projectile will exit the EMLS when it attains a velocity of 50 m/s (Fig. 4b). Figs. 4c and 4d show the predicted values of energy and power to be around 70 kJ and 694 kW respectively at the time of projectile exit.

Another study was done on the effect of variation of the parameters μ and b on the required value of the starting force F_{max} . As is observed from Fig. 5, the value of F_{max} does not vary significantly when the two coefficients are varied over their typical range of values.

TABLE 3: Design details of the SLIM

Parameter	Value
Primary iron core length	3000 mm
Primary iron core width	1000 mm
Secondary aluminium sheet width	3000 mm
Secondary back iron width	1000 mm
Air gap	10 mm
Secondary aluminium sheet thickness	3 mm
Secondary back iron thickness	6 mm
Slot width	25 mm
Tooth width	25 mm
Slot depth	25 mm
Number of slots	180
Conductors per slot	20
Current 70 A (rms),	99 A (peak)
Current density	1.1879×10^5 A/m
Number of poles	4
Power supply frequency	50 Hz
Stator winding arrangement	R,-B,Y, lap winding, double layer

III. DESIGN OF THE LIM

The next step is to design an LIM with a similar characteristic as obtained in Fig.4a. Parseval's method was used to determine the force-velocity characteristics of a single-sided LIM (SLIM). It is to be noted that the length of the SLIM (stator)

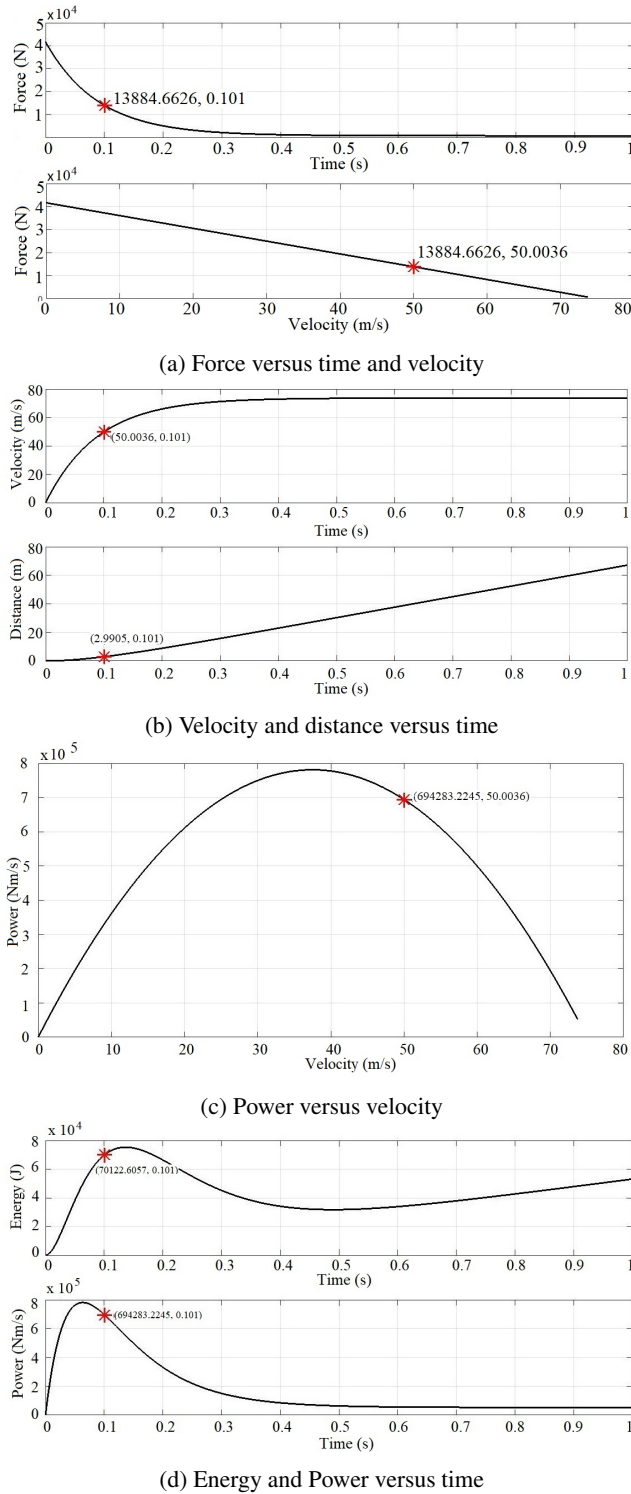


FIGURE 4: Plots for a launch angle of 30° . The red asterisk represents the instant when the projectile exits the launcher

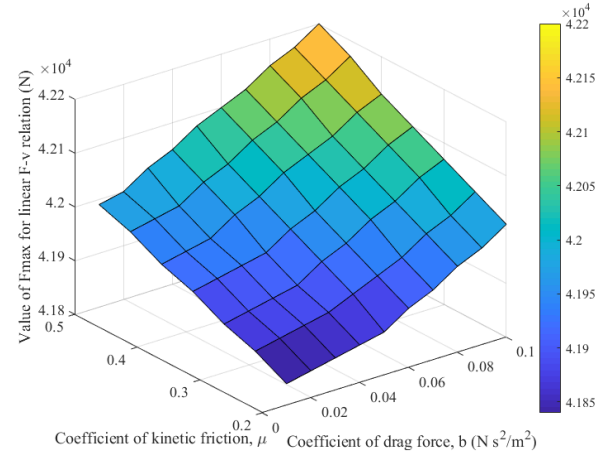


FIGURE 5: Surface plot of the values of the starting force F_{max} for different values of coefficient of kinetic friction μ and coefficient of drag force b

must be equal to the distance traveled by the projectile i.e. 3 meters

A. OPTIMIZING THE DESIGN PARAMETERS

It is well known that by increasing the secondary resistance of an induction motor, linear or rotary type, its torque or force-velocity curve becomes more and more inverse linear in shape.

In order to understand the effect of varying the design parameters on the performance characteristics of the SLIM, its parametric calculation was conducted based on Parseval's method. The nominal values of the parameters, i.e the values which are kept fixed while varying a parameter, unless stated otherwise, are shown in Table 3. From the calculation results obtained for constant-current drive, Fig. 6 and 7 demonstrates the manner in which thrust alters upon varying a parameter. By increasing primary iron width (Fig. 6a), marked rise in the peak thrust can be seen. Here, the secondary conductor width is taken to be the double of primary iron width in the parametric study. It can be inferred that while decreasing the primary iron width may make the characteristic's shape more suitable for the considered application (closer to inverse-linear in shape), the thrust value would also abate. It is to be noted that the ratio of secondary and primary width is kept fixed to be 2. Fig. 6b pertains to different secondary conductor widths and again the ratio of secondary and primary width is kept fixed to be 2. The peak thrust increases almost uniformly with the increasing secondary conductor width, while the starting thrust remains very much invariable like in (Fig. 6a). Fig. 6a and Fig. 6b differ on account of the Russel-Norsworthy correction factor for secondary conducting sheet resistivity being incorporated in case of the latter. The rotor resistivity increases upon decreasing the secondary conductor width. This implies higher secondary resistance, which explains the characteristics to become closer to inverse-linear in shape

for smaller secondary conductor widths. Notwithstanding the above, it may be remembered that in the case of constant voltage drive, the smaller width of primary will result in higher input current for identical input voltage and hence larger thrust. Furthermore, thrust multiplies with current square rule (Fig. 7a). Fig. 7d exhibits the behavior of the curves for different number of poles. Larger number of poles lead to greater thrust generated owing to the reduced pole pitches and thereby the synchronous velocities, for thrust falls off with increasing speed as the longitudinal (entry and exit end) effects become significant [23]. The thrust can also be made to surge by decreasing the air gap (Fig. 6d) (albeit the increase in thrust is slight when the air gap is reduced from 15 mm to 0.1 mm), supply frequency (Fig. 7c) (on account of the decrease in synchronous velocities) [24] and secondary conductive sheet thickness (Fig. 6c) (by the virtue of the reduced electromagnetic gap between the primary and the secondary iron). However, a thin layer of secondary sheet may melt and evaporate when large eddy currents will flow. As such a suitable thickness is to be selected complying Joule's limit criterion.

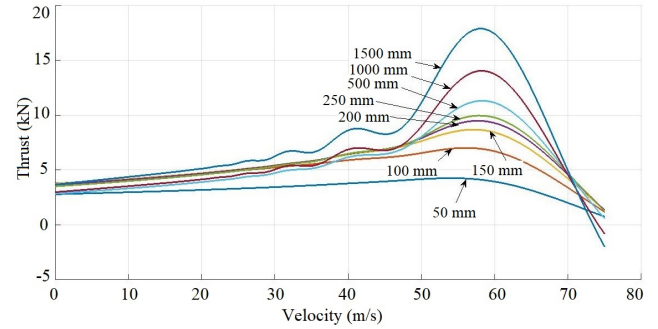
Fig.7b needs special inspection. Here, different metals have been evaluated for secondary sheet of conductivities in the order of $10^6 - 10^7 S/m$, with the highest conductivity of aluminum at $3.77 \times 10^7 S/m$ and the lowest being that of German Silver at $3.33 \times 10^6 S/m$. Within this range thrust increases with the lowering of the conductivity. Further lessening under this range would ensue a downturn in thrust as is indicated by the curve for carbon with conductivity of $2.88 \times 10^4 S/m$. It seems that German Silver is the most suitable candidate for electromagnetic launch. The width of the secondary was decided on the basis of Russell-Norsworthy correction factor for secondary overhang for sheet rotor machines. The Russell and Norsworthy formula [25] gives a correction factor for the effective secondary surface resistivity ρ'_r as

$$\rho'_r = \rho_r / (1 - k) \quad (3)$$

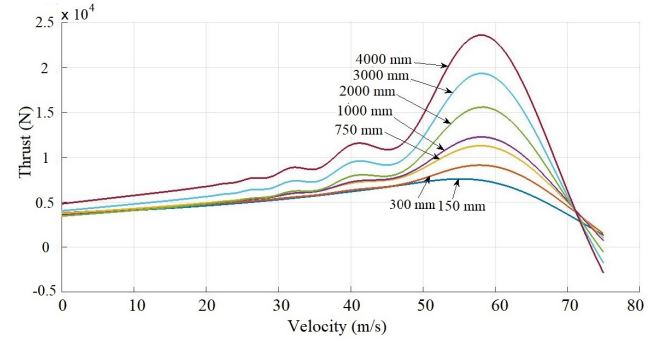
where $k = \tanh \epsilon / [\epsilon(1 + \tanh \epsilon \cdot \tanh \phi)]$, in which $\epsilon = \pi w / \tau$, $\phi = \pi \psi / \tau$, w is half of the stator width, τ is the pole pitch and ψ is the width of the overhang of the rotor plate beyond the edge of the stator. Fig. 8 shows the secondary surface resistivity versus secondary conductor width plots for fixed and variable primary iron widths. From the curves, it is observed that the rotor resistivity increases upon decreasing the secondary conductor width. The width of the secondary should be at least twice the width of the primary for it also decreases the transverse edge effect [26].

B. EXPERIMENTAL RESULTS FOR THRUST

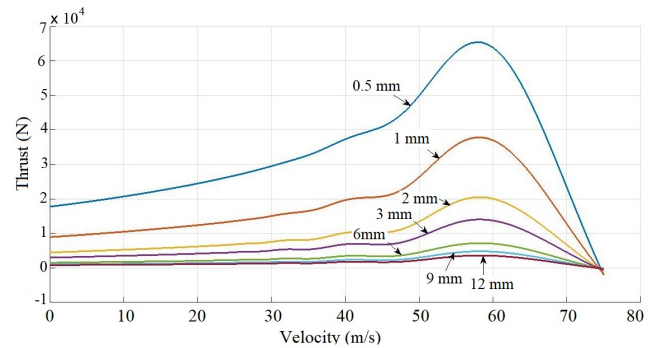
The fabricated static test rig of a smaller single sided linear induction motor (SLIM) is shown in Fig. 9. The design details are being given in Table 4. A prototype SLIM of smaller size than the proposed SLIM was fabricated and tested for different secondary conducting sheet materials to validate the performance characteristics of SLIM (Fig. 7b) which is one



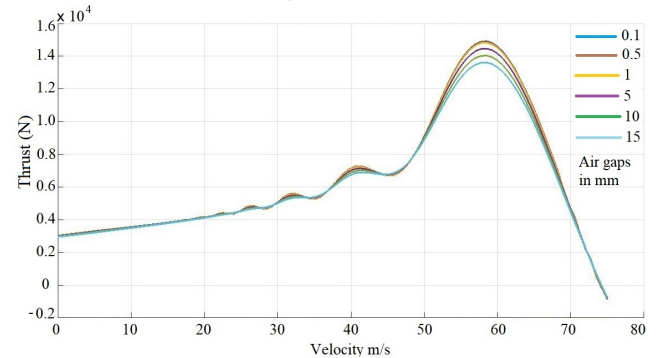
(a) Different primary iron widths



(b) Different secondary conductor widths



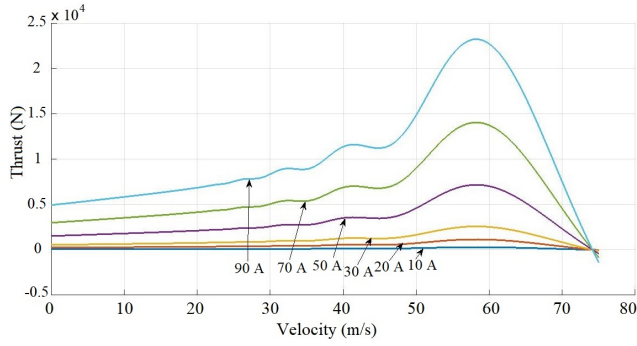
(c) Different secondary conductive sheet thicknesses



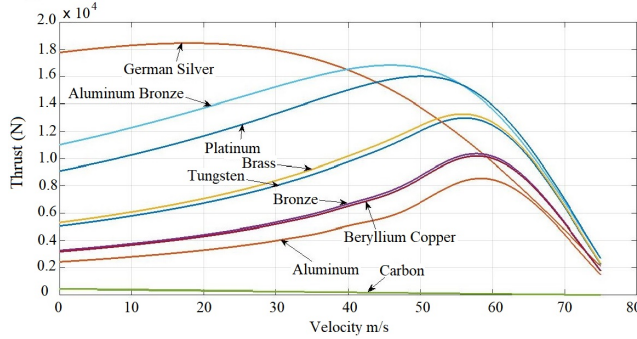
(d) Different air gaps

FIGURE 6: Performance characteristics of SLIM with varying design parameters.

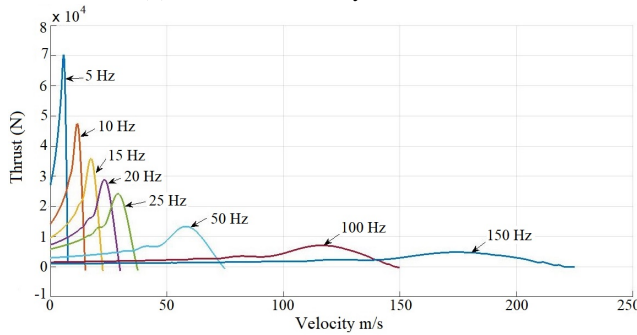
of our conclusive suggestion in the proposed work. Fig. 10 shows the VFD inverter, DSO for ascertaining the current, the stator and the three sheet secondary plates placed in a



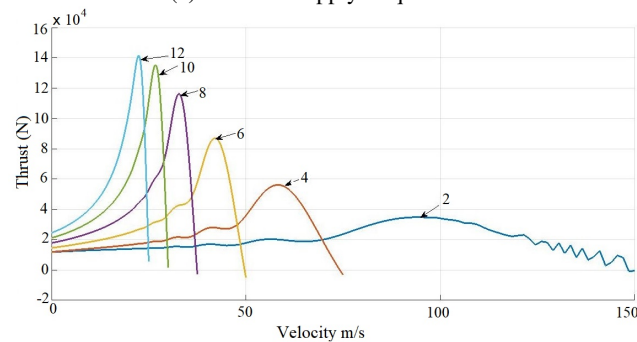
(a) Different currents



(b) Different secondary conductivities



(c) Different supply frequencies



(d) Different number of poles

FIGURE 7: Performance characteristics of SLIM with varying design parameters [continued].

staggered manner above the stator. The actual 3m SLIM is difficult to fabricate and test, owing to lack of finances. The sheet on the top is of aluminum with the back-iron glued above it, below of which is the sheet of German silver

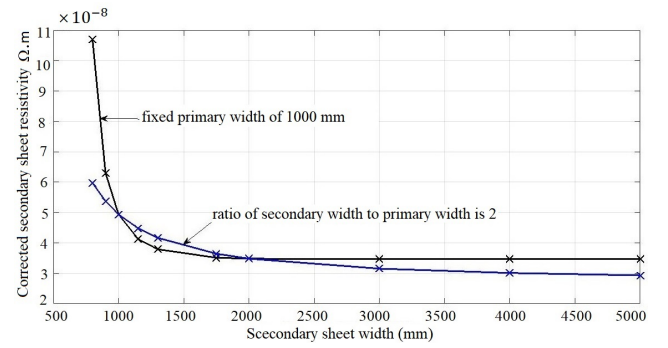


FIGURE 8: Effective resistivity variation due to secondary overhangs



FIGURE 9: Prototype SLIM stator with sliding support for secondary sheet



FIGURE 10: Setup for testing of the prototype SLIM

TABLE 4: Parameters of the fabricated SLIM

Parameter	Value
Primary iron core length (L , mm)	560
Secondary sheet length (L_s , mm)	750
Primary iron core width ($2w$, mm)	85
Secondary sheet width ($2d$, mm)	300
Secondary back iron width ($2d_i$, mm)	145
Air gap ($a - b$, mm)	2.2
Secondary sheet thickness ($b - c$, mm)	3
Secondary back iron thickness (c , mm)	4
Number of slots	24
Number of coils (Distributed 2 layer winding)	30
Number of coil sides per slot	2
Number of turns per coil	50
Coil pitch in terms of no. of slots	6
Pole pitch (τ , mm)	140
Number of poles	4
Power supply frequency (Hz)	50
Primary Current (A) rms	15

and the bottom sheet is of Beryllium copper. The fabricated LIMs have been theoretically analyzed using Fourier transform method (Parseval's) [22] pertaining to an equivalent SLIM. The Maxwell FEM software has been used to simulate and compute fields and forces. For standstill tests, variable frequency were obtained through 18.5 KW three phase ABB make VFD (ACS-600). For linear induction motor, the Force-speed characteristics and Standstill Force versus slip frequency characteristics are quite identical with certain tolerance because of the absence of end effects in standstill condition. In the static test rig, the standstill thrust at different slip frequencies for constant current was measured using digital spring balance. The heating of the secondary plate is avoided, because it alters the conductivity of plate. While Fig. 11 depicts the constant current (15A) normalized values measured and computed force of developed SLIM at different slip frequencies for aluminum as the secondary, Fig. 12 pertains to German Silver and Fig. 13 to Beryllium copper (conductivity of $2.865 \times 10^6 S/m$). The characteristics of aluminum and beryllium copper are quite identical. All the above three materials were of non-magnetic grade. The Finite Element and analytically computed forces are on the higher side due to the assumptions made and also because of slightly unbalanced three phase supply obtained during experimentation. It can be inferred that German Silver gives better thrust characteristics than aluminum as the secondary sheet material. It also has a higher melting point than that of aluminum.

C. SLIM DESIGN

The design parameters of the SLIM are chosen based on the study of parameter variation done and which are shown in Table 3. The above design of the SLIM is analyzed using Parseval's method and its performance characteristics are obtained (Fig. 14a and 14b). The force-velocity characteristic obtained in Fig. 14a needs to be amplified by a factor of about 4 in order to satisfy the conditions of the 3 m distance travel and 50 m/s exit velocity.

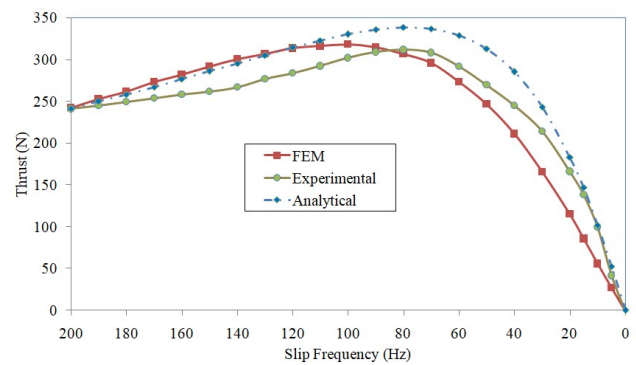


FIGURE 11: Thrust for SLIM with Aluminum as the secondary

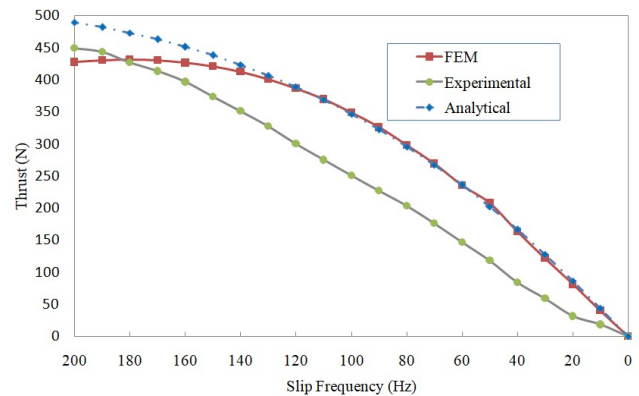


FIGURE 12: Thrust for SLIM with German Silver as the secondary

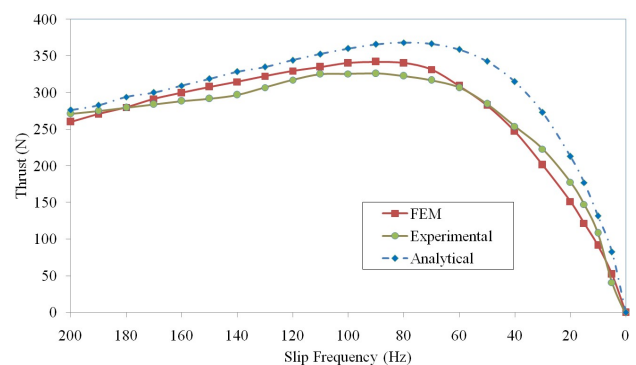
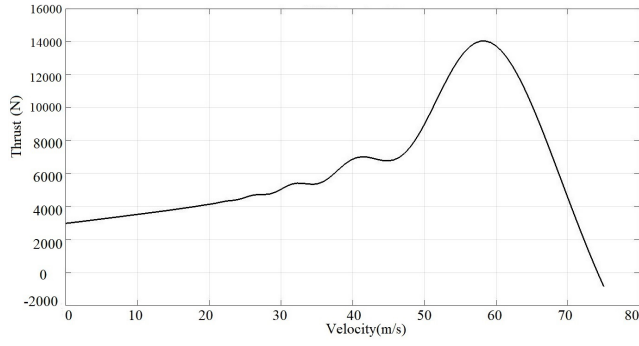
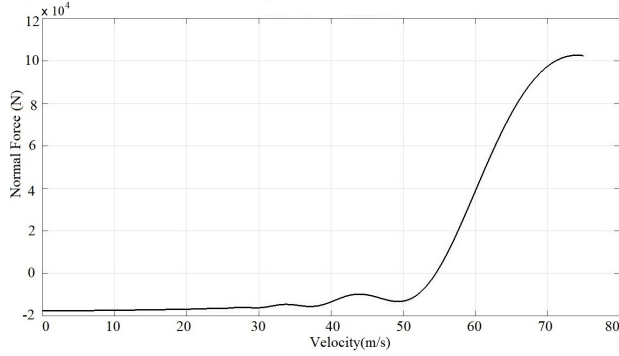


FIGURE 13: Thrust for SLIM with Beryllium copper as the secondary



(a) Thrust-velocity characteristic



(b) Normal force versus velocity curve

FIGURE 14: Performance characteristics of the designed SLIM. Gain required = 3.974.

This can be done by doubling the conductors per slot, as force is proportional to current density squared. The performance of the then designed SLIM for launch application was predicted by using the developed Simulink Model. In place of the inverse-linear relation between force and velocity, the actual characteristics of the designed SLIM are used for establishing the relation between force and velocity.

Fig. 15 shows the predicted performance of the designed SLIM-assisted EMLS, which seems to be poorer than that observed for the ideal case (Fig. 4). The projectile takes about 0.143 seconds to reach the velocity of 50 m/s in 3 meter travel. The energy and power values at the time of exiting the launcher are around 256 kJ and 1.7 MW respectively, both of which are considerably higher than the values obtained for the ideal case. This goes on to show how the design of the SLIM would affect its launch performance and power supply and cooling requirements.

D. MODEL SLIM DESIGN

A 36 cm-long model SLIM (Fig. 16) was also designed having the following parameters as detailed in (Table 5).

The model SLIM was modeled and simulated in ANSYS 3D Maxwell software and its performance characteristics were obtained. The same was also verified using Parseval's method (Fig. 17).

TABLE 5: Design details of the Model SLIM

Parameter	Value
Primary iron core length	360 mm
Primary iron core width	100 mm
Secondary aluminium sheet width	250 mm
Secondary back iron width	100 mm
Air gap	15 mm
Secondary aluminium sheet thickness	2 mm
Secondary back iron thickness	300 mm
Slot width	8 mm
Tooth width	7 mm
Slot depth	10 mm
Number of slots	24
Conductors per slot	50
Current 49.5 A (rms),	70 A (peak)
Current density	2.33×10^5 A/m
Number of poles	4
Power supply frequency	50 Hz
Stator winding arrangement	R,-B,Y, lap winding, double layer

IV. THERMAL CHARACTERISTICS OF THE LIM

In EM launch application, large amount of energy flows through the LIM for a short duration of time. In such condition, it becomes important to predict the thermal performance of the machine. This section describes the thermal modeling of the LIM using heat balance equation. Also, ANSYS Maxwell and Mechanical are coupled and the model is solved for temperature distribution using FEM.

A. THERMAL MODEL

Inherently, electrical machines consist of different parts made up of different materials like copper, iron and aluminum. By following a lumped parameter approach, it is possible to estimate the temperature rise in any part of the machine, the underlying assumption being that the temperature distribution is uniform over the surface of the body.

Let P_1 = heat developed, W

P_2 = heat dissipated to the cooling medium, W

W = weight of the active part of the machine, kg

h = specific heat, J/kg/°C;

A = cooling surface area, m^2

d = coefficient of heat transfer, J/sec/ m^2 /°C; and

θ = mean temperature rise, °C.

During dt , let the machine temperature rise by $d\theta$. Then,

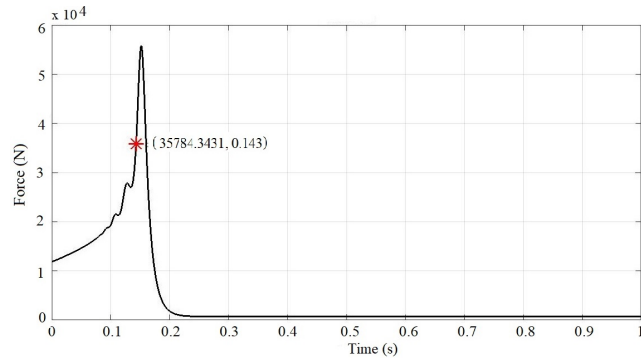
$$\text{Heat absorbed} = \text{Heat developed} - \text{Heat dissipated}$$

$$W.h.d\theta = P_1.dt - P_2.dt$$

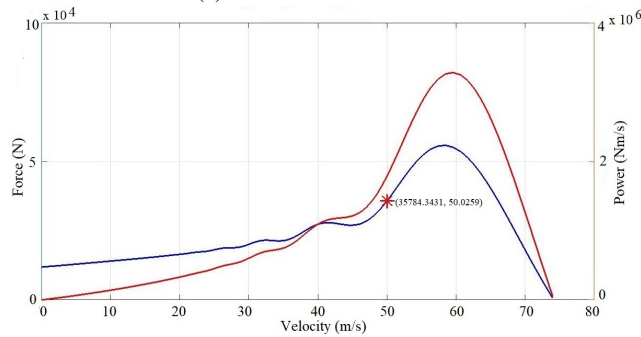
$$W.h.d\theta = P_1.dt - \theta.dA.dt$$

$$\frac{d\theta}{dt} + \frac{D}{C}\theta = \frac{P_1}{C} \quad (4)$$

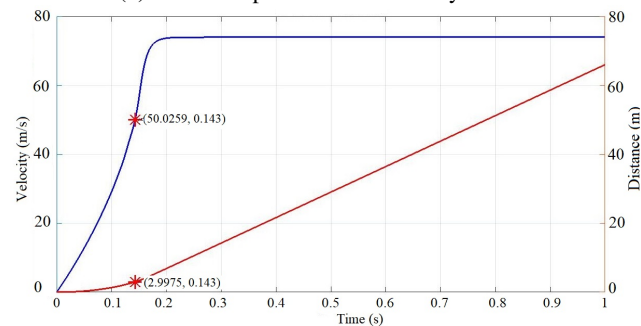
where $C = W.h$ is the thermal capacity of the machine in W/°C and $D = d.A$ is the heat dissipation constant in W/°C. Using the heat balancing Eq. (4), the temperature distribution in any part of the machine can be determined [27]. Eq. (4)



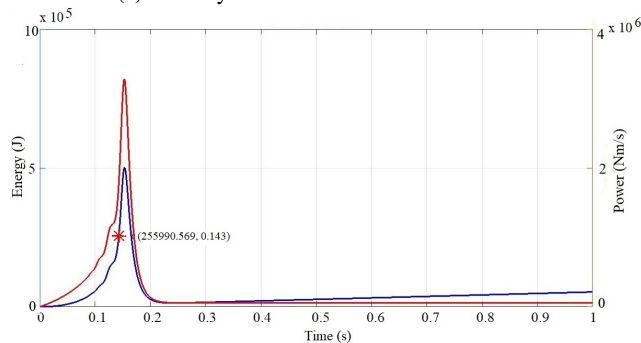
(a) Force versus time curve



(b) Force and power versus velocity curves



(c) Velocity and distance versus time curves



(d) Energy and power versus time curves

FIGURE 15: Plots for launch using the designed SLIM. The red asterisk represents the instant when the projectile exits the launcher.

can be solved for temperature rise using Laplace transform

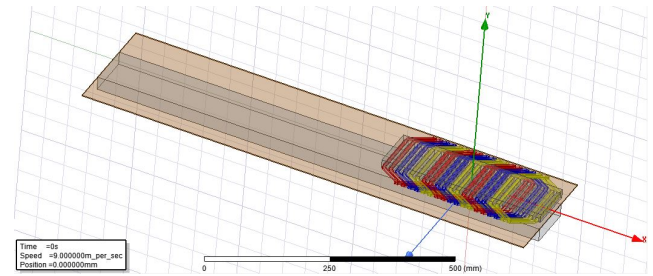
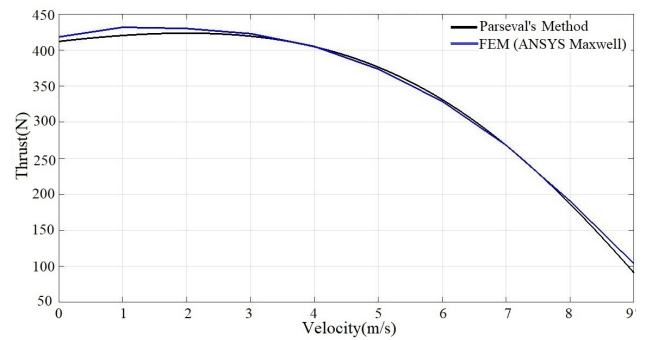
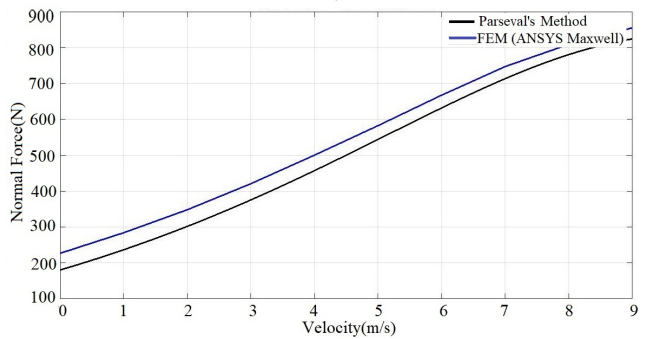


FIGURE 16: 3D model of the model-SLIM



(a) Thrust-velocity characteristic



(b) Normal force-velocity characteristics

FIGURE 17: Performance characteristics of the model SLIM obtained by the two methods

and its expression during heating is obtained as

$$\theta(t) = \theta_m(1 - e^{-\frac{t}{\tau}}) + \theta_o.e^{-\frac{t}{\tau}} \quad (5)$$

where θ_o is the initial temperature in $^{\circ}\text{C}$, $\theta_m = P_1/D$ is the steady-state temperature rise in $^{\circ}\text{C}$ and $\tau = C/D$ is the heating time constant in $/\text{sec}$.

B. TWO-WAY COUPLING

In order to determine the temperature distribution in the LIM in ANSYS, the calculated losses in Maxwell are exported to ANSYS Mechanical. This coupling between these two softwares enables only one-way transfer of data. The thermal analysis data can also be fed back to the EM solver, which then updates the properties of the materials based on the imported temperature distribution and re-solves the model

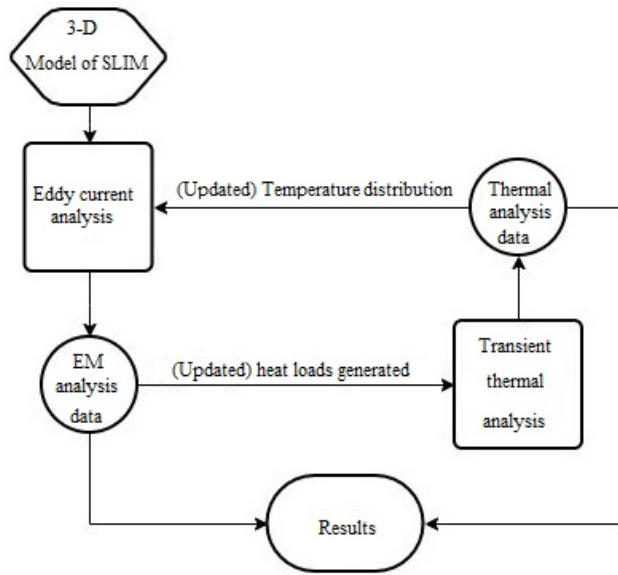


FIGURE 18: Flowchart of two-way coupling between ANSYS/Maxwell and Mechanical/Thermal

and generates new loss distribution. Such an analysis is called a two-way coupling and is described in Fig. 18.

Such an analysis describes the real world case where the properties of the materials such as bulk conductivity vary with temperature, resulting in non-uniform distribution of heat. The iterative process of the two-way coupling can be continued until the solution stops changing within a desired level of tolerance.

V. RESULTS

The designed SLIM model was solved for the temperature distribution in its coils and the aluminium sheet for standstill condition. The simulation was performed for the heat generated being applied to the bodies for 1 second, though it must be noted that the projectile exits the launcher within 0.15 seconds. The results obtained from the analytical thermal model and the FEM software are compared.

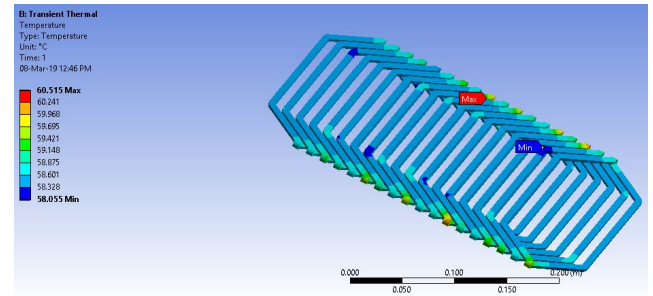
A. MODEL SLIM

Fig. 19 shows the results obtained for the designed model SLIM. In the FEM Simulation the temperature rise in the coils is about 38°C (from 22°C to 60°C), which is close to the value predicted by the analytical model.

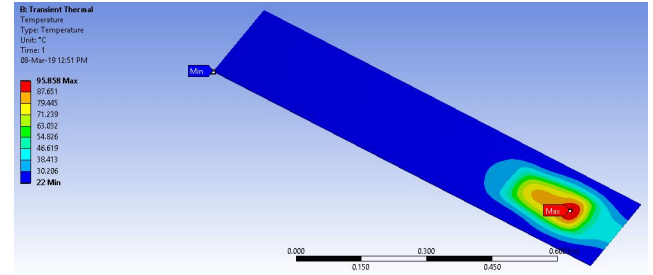
However, for aluminum sheet, the distribution of temperature turns out to be very non-uniform in the simulation, the highest temperature reached being around 95°C . The analytical model performs badly for this case and predicts the maximum temperature of the aluminum sheet to be 30°C after 1 second.

B. LONG SLIM

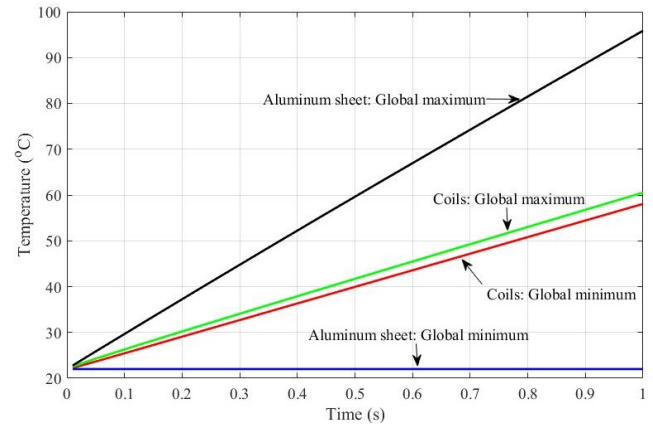
Two-way coupling between the EM and thermal solvers was employed when the long SLIM was analyzed in ANSYS



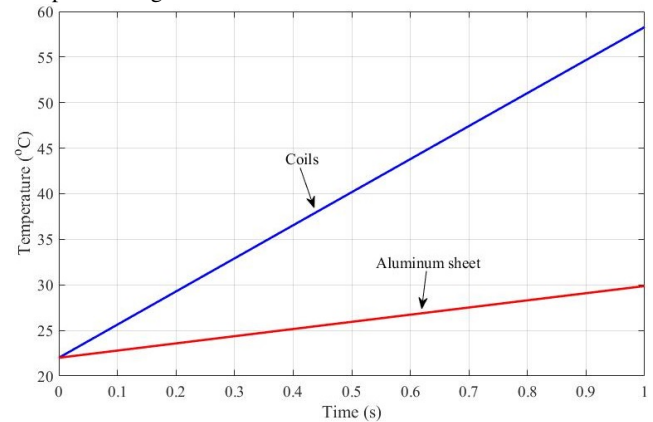
(a) Temperature distribution in the coils of the model SLIM



(b) Temperature distribution in the aluminum sheet of the model SLIM

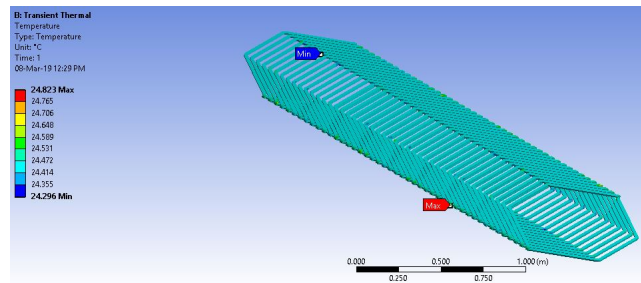


(c) Temperature of the coils and aluminum sheet of the model SLIM computed using ANSYS Workbench

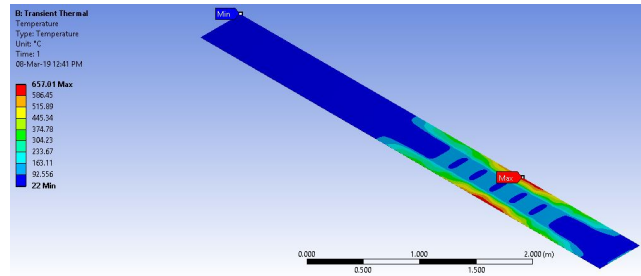


(d) Temperature of the coils and aluminum sheet of the model SLIM computed analytically

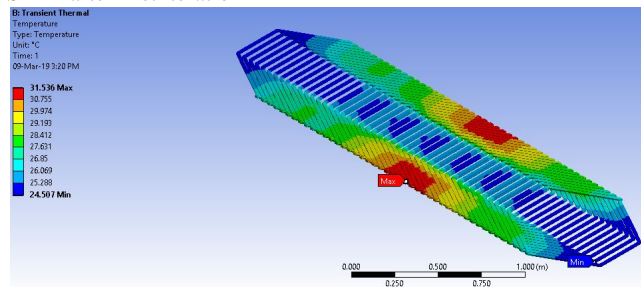
FIGURE 19: Results of thermal analysis of model SLIM



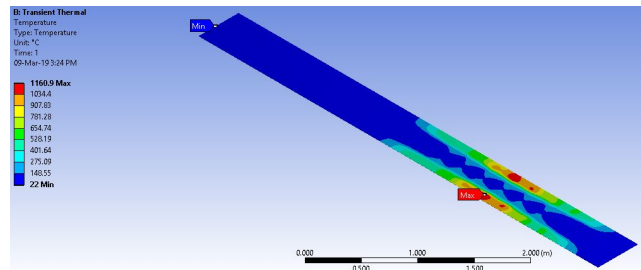
(a) Temperature distribution in the coils of the long SLIM after first iteration



(b) Temperature distribution in the aluminum sheet of the long SLIM after first iteration



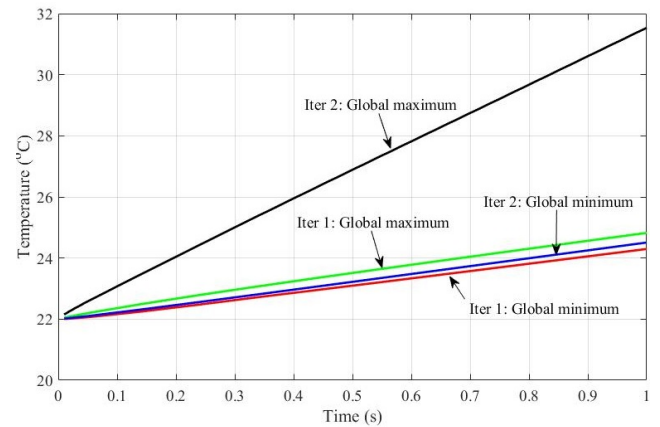
(c) Temperature distribution in the coils of the long SLIM after second iteration



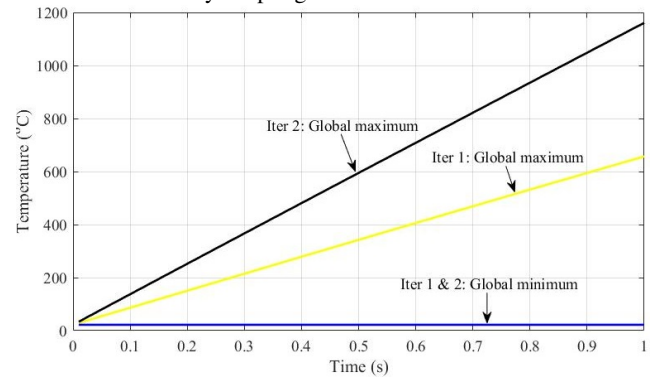
(d) Temperature distribution in the aluminum sheet of the long SLIM after second iteration

FIGURE 20: Results of thermal analysis of long SLIM

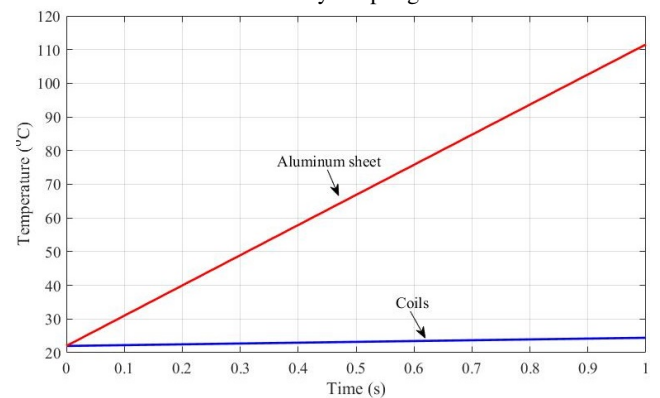
Workbench. Fig. 20 shows the temperature distribution in the coils and the aluminum sheet after the first and second iteration of the simulation after 1 second. It is evident from the results that the temperature dependence of the bulk conductivities of materials such as copper and aluminum strongly affect their temperature distribution. After the first iteration, the temperature distribution is almost uniform in the coils, and hence, the analytical model predicts its temperature with high accuracy (Fig. 21).



(a) Temperature of the coils of the long SLIM in the first and second iterations of two-way coupling



(b) Temperature of the aluminum sheet of the long SLIM in the first and second iterations of two-way coupling



(c) Temperature of the coils and aluminum sheet of the long SLIM computed analytically

FIGURE 21: Comparison of results of thermal analysis of long SLIM

The analytical model fails for the aluminum sheet, as was observed in the model SLIM case too. After the second iteration of the simulation, the temperature after 1 second increases for both the coils and the sheet. Moreover, the temperature distribution has become non-uniform for the coils too. It is because of this non-uniformity and the inability of the model to adapt itself with changes in conductivities that

it fails to predict the temperature values with accepted level of accuracy. Another important aspect of temperature rise in the machine can be observed by comparing Fig. 19a and 20a. While the model SLIM is much shorter in length than the long SLIM, the surface area available for the coils to dissipate the heat generated in them is higher for the long SLIM. The model SLIM has a total power loss of 21.24 kW in its winding, whereas the long SLIM has that value equal to 182.44 kW (after first iteration). Still, the temperature rise is greater for the model SLIM. This observation can also be made by making the derivative term in Eq. (4) equal to zero (steady-state). Then, the steady-state temperature rise, $\theta_m = P_1/D = P_1/(d.A)$, which shows that the final steady temperature rise is directly proportional to the power loss and inversely proportional to the cooling surface area. Thus, for the same power loss, the machine having smaller dissipating surface area or poor ventilation would attain higher temperature rise.

VI. CONCLUSION

The main conclusions and suggestions drawn on the basis of analysis, investigation and experimentation are enumerated as follows:

- 1) Designing an SLIM for EM launch application involves shaping its thrust-velocity curve to be inverse-linear which is in itself a challenging task. By reducing the secondary conductive sheet thickness and increasing the air clearance, the characteristic seems to become closer to what is desired but too thin of the secondary sheet might result in its melting. Also, larger air gap means more magnetizing current drawn from the supply to deliver the same power output, implying increased losses and hence temperature rise. German silver (non-magnetic grade) has been identified as the most suitable material for the secondary conductive sheet of the SLIM for EMLS because its thrust-velocity curve is close to inverse linear relationship (stable region of operation of LIM), which significantly reduces the accelerating time during the launch. Further, alternative materials such as zirconium-copper alloy can be opted for the primary winding wire and secondary conductor which have higher melting points and can withstand greater thermal stresses.
- 2) The analytical model for thermal simulation of LIM follows a lumped approach and fails to capture the many practicalities and non-idealities of temperature distribution. To overcome this limitation, either this simple model can be further refined to define multiple lumps in the system or an FEM-based simulation can be conducted to accurately predict the temperature rise in the aluminum sheet, which is presented in the work.
- 3) The study reports constant current and frequency design of EMLS. It has been observed that constant thrust versus speed of launch for the same exit velocity yields lesser thrust requirements. However, it requires the Variable Voltage Variable Frequency (VVVF) technology for its operation and also, the launch is a bit

delayed. Furthermore, the mechanical time constant of such energy machines are comparable to its electrical time constant, rendering VFD control difficult to implement.

- 4) The study indicates the need of either increasing the supply frequency or going for a multi-section stator, with each section having progressively increasing frequency by employing Rising Frequency Generator (RFG). A variable pole pitch LIM is also suggested with compensating windings which overcomes the design challenge.

The forthcoming advanced work is undertaken as follows:

- 1) These machines are subjected to very high voltage 3-phase pulse power. The presence of unstable zone (positive slope) in the transient thrust-speed characteristics, oscillations in thrust may occur during the launch. The analysis of the transient current and dynamic thrust to assess the stability of the launch is being undertaken.
- 2) The experimental assessment of Thermal aspects of this research work is proposed for the future work by envisaging heat sensors to be employed for recording the surface temperatures of the stator and coils of the prototype SLIM.
- 3) In the present work, double layer, full pitch winding has been assumed. Several other winding configurations are also envisaged like single layer winding and tooth (polar) windings for obtaining optimum operation.

Further studies involve the EMI/EMC aspects of such machines.

APPENDIX PARSEVAL'S METHOD OF FOURIER TRANSFORM APPLICABLE FOR LINEAR INDUCTION MOTOR

The following symbols are defined for the different parameters of the SLIM as shown in Fig. 22:

- L : Primary stator length
- w : Half width of Primary iron (stator core)
- d : Half width of Secondary conductor plate
- $a - b$: Air gap
- $b - c$: Thickness of secondary conductive sheet
- c : Thickness of secondary backiron
- J_m : Primary current density
- P : Number of poles
- τ : Pole pitch
- σ_1 : Conductivity of iron
- σ_2 : Conductivity of secondary conductive material
- μ_0 : Permeability of air
- μ_1 : Permeability of iron
- μ_2 : Permeability of secondary conductive material
- n : order of harmonic representing transverse variation of field by means of Fourier series
- ξ : The Fourier space Transform counterpart of x-coordinate, or wave number in per meters

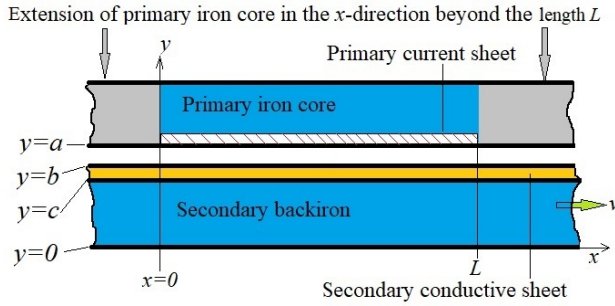


FIGURE 22: Model of single-sided linear induction motor [22]

Then, the thrust of the SLIM is given by:

$$F_t = \frac{\mu_3 J_m^2}{\pi} \int_{-\infty}^{\infty} \frac{1 - \cos(\xi + k)L}{\xi(\xi + k)^2} \cdot \sum_n c_n \frac{\eta_n}{\lambda_n} \Im \left[\frac{W_n(\xi, a)}{K_n(\xi)} \right] \sin \lambda_n w d\xi \quad (6)$$

where,

$$J_m = \frac{\sqrt{2} \times \text{Slots} \times \text{Cond. per slot} \times \text{Current}}{L} \quad (7)$$

$$k = \frac{\pi}{\tau} \text{ Wave constant, per meter} \quad (8)$$

$$\lambda_n = \frac{n\pi}{2d}, n = 1, 3, 5... \quad (9)$$

$$c_n = \frac{2 \sin \lambda_n w}{\lambda_n d} \quad (10)$$

$$\eta_n = \sqrt{\xi^2 + \lambda_n^2} \quad (11)$$

$$\chi_n = \sqrt{\xi^2 + \lambda_n^2 + j\omega\mu_1\sigma_1 + j\xi v\mu_1\sigma_1} \quad (12)$$

$$\gamma_n = \sqrt{\xi^2 + \lambda_n^2 + j\omega\mu_2\sigma_2 + j\xi v\mu_2\sigma_2} \quad (13)$$

λ_n is the index number of the Fourier series in the transverse direction

c_n is the coefficient of the Fourier series expansion of the rectangle, which represents the primary current in the transverse direction

η_n , χ_n and γ_n are the Fourier layer indices of air, backiron and secondary sheet respectively.

$$K_n(\xi) = \cosh \eta_n(a - b) \cosh \gamma_n(b - c) \left\{ 1 + \frac{\mu_0 \chi_n}{\mu_1 \eta_n} \tanh \chi_n c \right\} + \sinh \eta_n(a - b) \cosh \gamma_n(b - c) \left\{ 1 + \frac{\mu_1 \eta_n}{\mu_0 \chi_n} \tanh \chi_n c \right\} + \cosh \eta_n(a - b) \sinh \gamma_n(b - c) \left\{ \frac{\mu_0 \gamma_n}{\mu_2 \eta_n} + \frac{\mu_1 \gamma_n}{\mu_2 \chi_n} \tanh \chi_n c \right\} + \sinh \eta_n(a - b) \sinh \gamma_n(b - c) \left\{ \frac{\mu_2 \eta_n}{\mu_0 \gamma_n} + \frac{\mu_2 \chi_n}{\mu_1 \gamma_n} \tanh \chi_n c \right\} \quad (14)$$

$$W_n(\xi, y) = \cosh \eta_n(y - b) \cosh \gamma_n(b - c) \cdot \left\{ 1 + \frac{\mu_1 \eta_n}{\mu_0 \chi_n} \tanh \chi_n c \right\} + \sinh \eta_n(y - b) \cosh \gamma_n(b - c) \left\{ 1 + \frac{\mu_0 \chi_n}{\mu_1 \eta_n} \tanh \chi_n c \right\} + \cosh \eta_n(y - b) \sinh \gamma_n(b - c) \left\{ \frac{\mu_2 \eta_n}{\mu_0 \gamma_n} + \frac{\mu_2 \chi_n}{\mu_1 \gamma_n} \tanh \chi_n c \right\} + \sinh \eta_n(y - b) \sinh \gamma_n(b - c) \left\{ \frac{\mu_0 \gamma_n}{\mu_2 \eta_n} + \frac{\mu_1 \gamma_n}{\mu_2 \chi_n} \tanh \chi_n c \right\} \quad (15)$$

Also, the total attracting force is given by:

$$F_p = F_{px} + F_{py} + F_{pz} \quad (16)$$

where,

$$F_{px} = -\frac{\mu_0 J_m^2 d}{4\pi} \sum_{n=1}^{\infty} c_n^2 \int_{-\infty}^{\infty} \frac{1 - \cos \xi L}{(\xi + k)^2} d\xi \quad (17)$$

$$F_{py} = \frac{\mu_0 J_m^2 d}{4\pi} \int_{-\infty}^{\infty} \sum_{n=1}^{\infty} c_n^2 \left[\frac{1 - \cos \xi L}{\xi^2(\xi + k)^2} \eta_n^2 \left| \frac{W_n(\xi, y)}{K_n(\xi)} \right|^2 + \frac{1 - \cos \xi L}{\xi^2} \frac{\lambda_n^2}{k^2} \left| \frac{W_n(0, y)}{K_n(0)} \right|^2 - \frac{\eta_n \lambda_n}{k} \frac{2(1 - \cos \xi L)}{\xi^2(\xi + k)} \Im \left\{ \frac{W_n(\xi, y)}{K_n(\xi)} \frac{W_n(0, y)}{K_n(0)} \right\} \right] d\xi \quad (18)$$

$$F_{pz} = -\frac{\mu_0 J_m^2 d}{4\pi} \sum_{n=1}^{\infty} \lambda_n^2 c_n^2 \int_{-\infty}^{\infty} \left(\frac{1 - \cos \xi L}{\xi^2(\xi + k)^2} \right) \cdot \left(+ \frac{1 - \cos \xi L}{\xi^2 k^2} + \frac{2(1 - \cos \xi L)}{k \xi^2(\xi + k)} \right) d\xi \quad (19)$$

The method of performing the numerical integration can be found in [28].

ACKNOWLEDGMENT

The authors are thankful to the Government of India for Project Support Grant against Project No. R&D/X X X X/EE/18-19/02.

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